

# HEAT PUMPS

*DIDACTIC MATERIALS*

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Virtual and Intensive Course  
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- In heat pump system does not use gas, it uses substance, named the service medium.
- Service medium is characterised by the fact that it can be in all three aggregate states.
- Aggregate, phase transition has an extremely high energy density, it relatively easily allow the process to be approximated to the Carnot process.
- High energy density allows more compact heat pump systems to be produced.
- Approximation to the Carnot process is because substances as a phase transition is always an isothermic process.

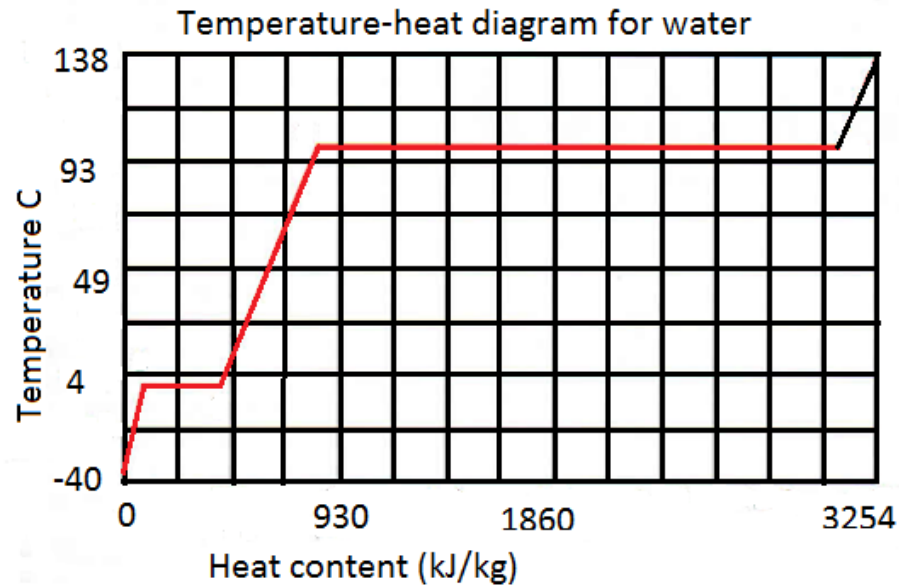


Fig.1. Temperature-heat diagram for water

## The heat pump in the log p, h diagram

- In heat pump engineering, it is simple to represent the cycle in a log p, h diagram.
- In log p, h diagram energy differences easily identified.
- Every service medium (refrigerant) has its own diagram depending on its thermodynamic characteristics.

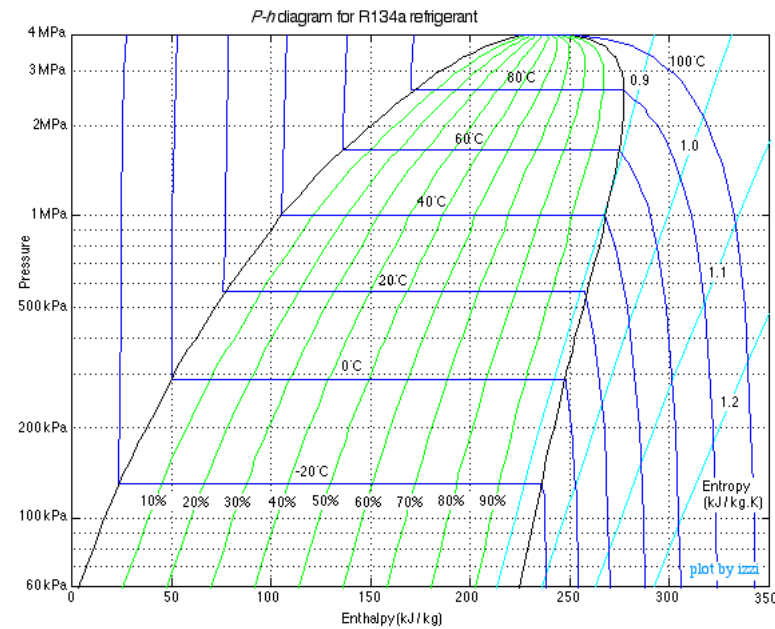


Fig.2. P-h diagram for R134a refrigerant

# Operating principle of the heat pump

- The fundamental principle of the heat pump cycle is the dependency of the boiling temperature on pressure.
- Heat absorption must be at a lower temperature and heat emission at a higher temperature.

- This is reached by evaporating the refrigerant at a low temperature to take the necessary heat flow from the heat sources.
- To make this process, the pressure must be reduced so that the evaporation temperature is below the desired heat sources temperature.



# Operating principle of the heat pump

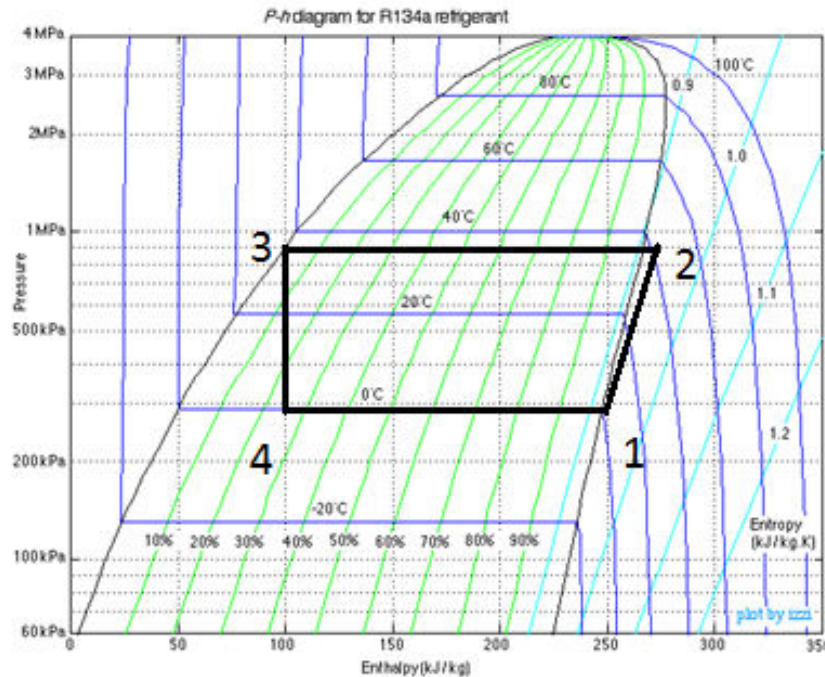


Fig.3. The heat pump cycle in a p-h diagram

- Line 1-2 Fig.3, compressing the refrigerant to a higher pressure, at which the boiling temperature is above the user's temperature.
- Line 2-3 Fig.3 condensation of the refrigerant in the condenser, the heat flow  $Q_{out}$  is discharged to the user.
- Line 3-4 Fig.3 throttling, an expansion valve is used for throttling .
- Line 4-1 Fig.3 Evaporation. During evaporation, the refrigerant absorbs the heat flow  $Q_{in}$  from the heat sources.

$Q_{in}$ - heat pump heat supplied from heat source.



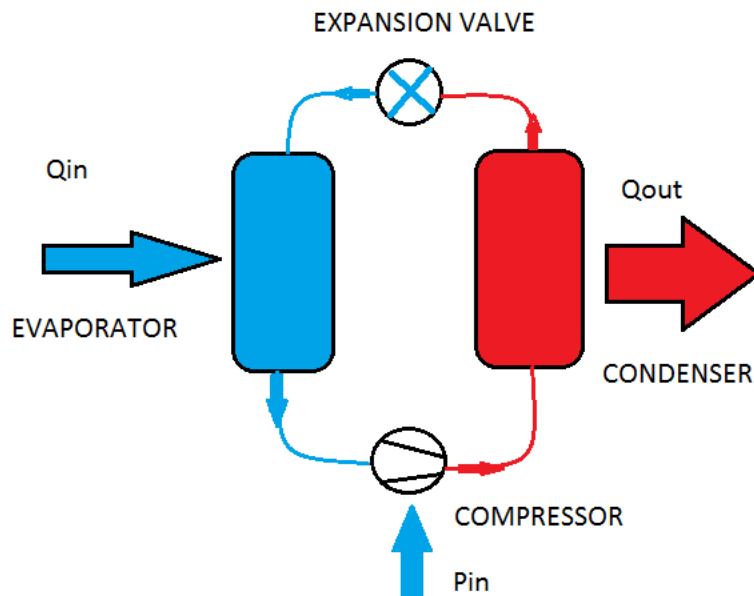


Fig.4. The heat pump four main components

- **Compressor** – used to raise refrigerant temperature.
- **Condenser** – used to transfer heat from refrigerant to the heat user.
- **Expansion valve** – used to reduce refrigerant temperature.
- **Evaporator** – used to transfer heat from the heat source to refrigerant.

## Functioning of a compression heat pump system

- The task of a compression heat pump system, is to transport heat energy  $Q_{in}$  from a low temperature level to a high temperature level ( $Q_{out}$ ).
- In the picture Fig.4 of a heat pump, ( $P_{in}$ ) supplied mechanical energy to the system from outside by a compressor.

## Functioning of a compression heat pump system

- System must satisfy the 1st law of thermodynamics

$$Q_{out} = Q_{in} + P_{in}$$

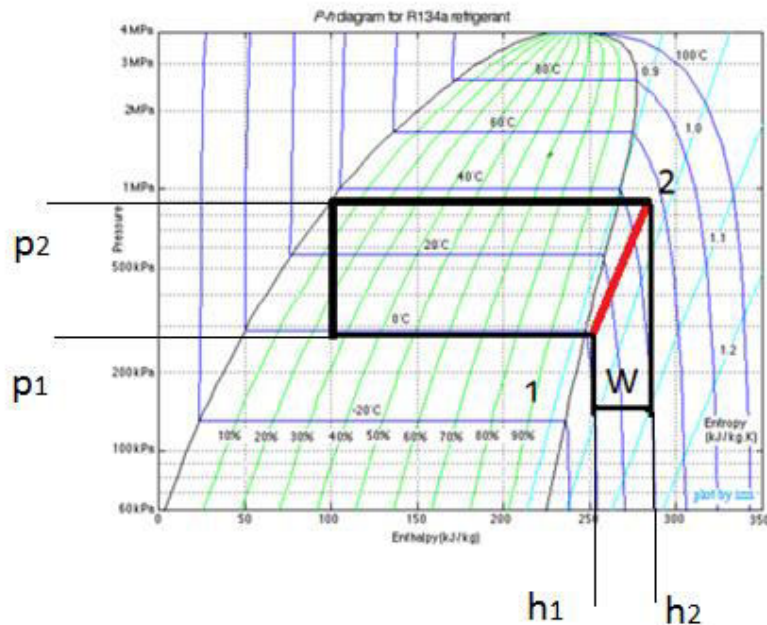
Where:

$Q_{out}$  – output of the evaporator (heat pump heat supplied to the user).

$Q_{in}$  – output of the condenser (heat pump heat supplied from heat sources).

$P_{in}$  – compressor output (supplied compressor mechanical energy).

# CALCULATIONS USING THE (p-h) DIAGRAM



**1st stage** – Fig. 5 Isentropic compression (1-2) – the refrigerant gaseous taken in by the evaporator is compressed from the evaporation pressure  $p_1$  to the condensation pressure  $p_2$  by a compressor.

$p_1$  – compressor inlet pressure.

$p_2$  – outlet pressure of the compressor.

Fig. 5. The compressor work in p-h diagram

- In ideal conditions, compression is isentropic.
- In real compression process heat caused by friction.
- Heat will be transferred through the cylinder wall.
- The enthalpy increase equivalent to the amount of work used by compressor.

- The compressor output:

$$P_v = (h_2 - h_1) \cdot m_R \quad (\text{kW})$$

- The specific compressor work:

$$W = P_v / m_R = h_2 - h_1 \quad (\text{kJ/kg})$$

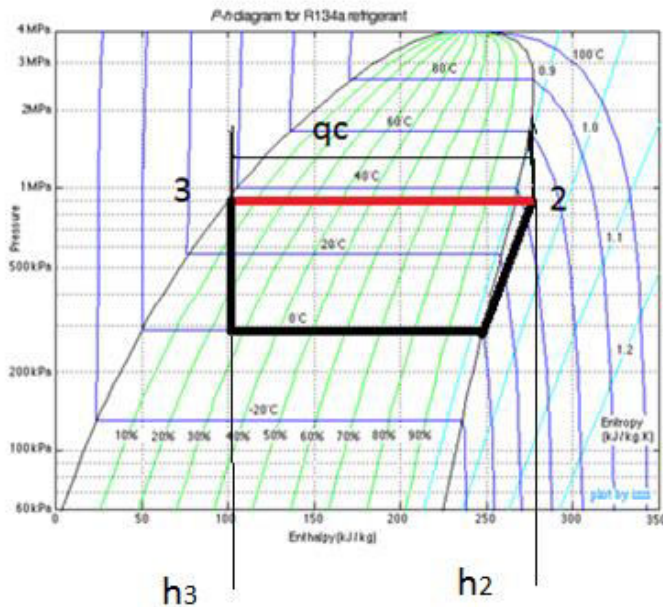
where:

$h_1$  – compressor inlet enthalpy (kJ/kg)

$h_2$  – compressor outlet enthalpy (kJ/kg)

$m_R$  – freon mass flow rate (kg/s)





- **2nd stage** – Fig.6 Isobaric condensation (2-3) - heat energy is released to the environment by condensation of the refrigerant.
- There are the heat flow absorbed on the evaporator and the power intake of the compressor.

Fig.6. The condensation process in p-h diagram



- The total output of the condenser:

$$Q_c = (h_2 - h_3) \cdot m_R \text{ (kW)}$$

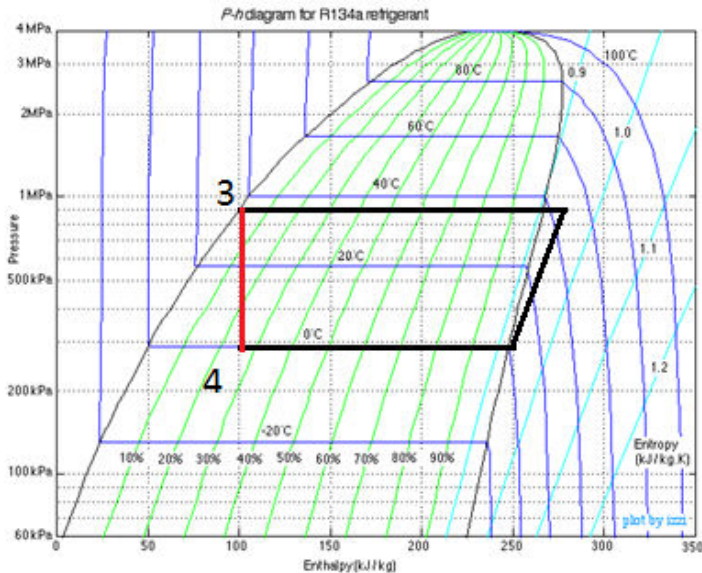
- The specific condenser output:

$$q_c = Q_c / m_R = h_2 - h_3 \text{ (kJ/kg)}$$

where:

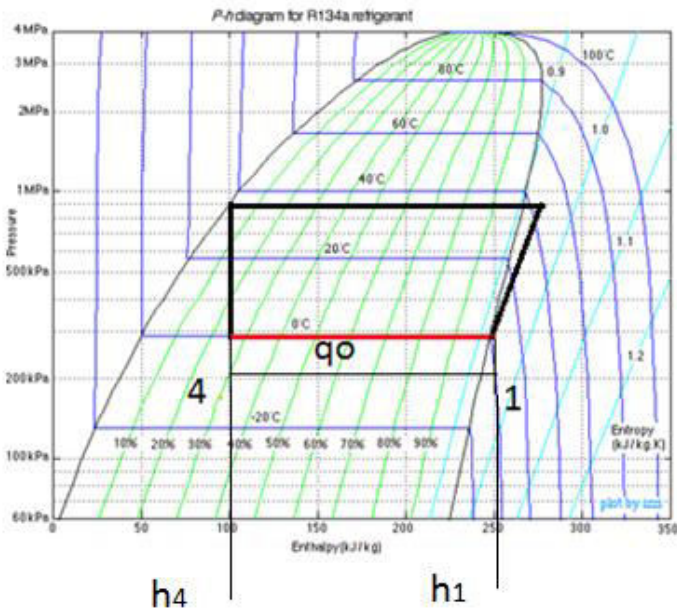
$h_2$  – condenser inlet enthalpy (kJ/kg)

$h_3$  – condenser outlet enthalpy (kJ/kg)



- **3rd stage-** Fig.7. Isenthalpic throttling (3-4).
- Completely condensed refrigerant is supplied to the evaporator at low pressure.
- This take place during an isenthalpic throttling.
- During process neither heat nor work is exchanged with the environment.

Fig.7. The throttling process in p-h diagram



- **4th stage-** Fig.8. Isobaric evaporation.
- The liquid refrigerant enters the evaporator after throttling, where it evaporates.
- Evaporation consumes heat source heat.

Fig.8. The evaporation process in p-h diagram

- The total output of the evaporator can be calculated:

$$Q_o = (h_1 - h_4) \cdot m_R \quad (\text{kW})$$

- The specific freon capacity is calculated as:

$$q_o = Q_o / m_R = h_1 - h_4 \quad (\text{kJ/kg})$$

where:

$h_4$  – evaporator inlet enthalpy (kJ/kg)

$H_1$  – evaporator outlet enthalpy (kJ/kg)

- The heat pump output coefficient:

$$\varepsilon = q_c/w = q_c/(q_c - q_o) = Q_c/P_v$$

where:

$\varepsilon$  – coefficient of performance

$q_c$  – specific condense capacity (kJ/kg)

$q_o$  – specific evaporator capacity (kJ/kg)

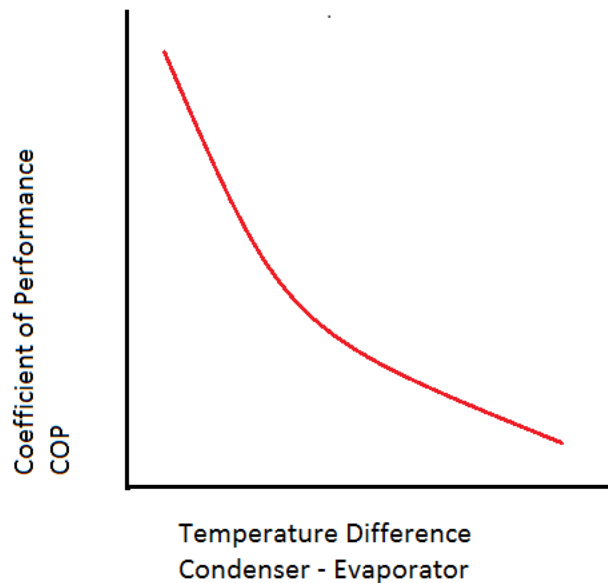


Fig. 9. Coefficient of performance depending on evaporation and condensation temperature

- The coefficient of performance is determined by the evaporation and condensation temperature Fig. 9.
- The coefficient of performance is higher, the less the evaporation and condensation temperature difference.
- The maximum coefficient of performance is given by the Carnot process.



## Components of a compression refrigeration system:

- **The compressor** – one of the four main components of a heat pump system. Its task is to supply circuit mechanical energy.
- The compressor takes gaseous refrigerant at low pressure, compresses and emits it at high pressure.
- During compression, the temperature of the gas increase.



There are two basic compressor types:

- **Displacement engines**, the working chamber is periodically filled and drained. The most widespread type of displacement engine is the piston compressor.

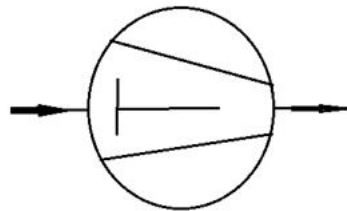


Fig. 10. Symbol for a piston compressor

## Dynamic type compressors:

The working medium is supplied with kinetic energy, and this is then transformed into pressure energy.

- Dynamic type compressors allowing very large flow volumes.
- Dynamic type compressors are used for a compressor output of more than around 400kW.

## Capacity of piston compressor

The capacity of a system depends on the evaporation temperature.

- The capacity decreases as the evaporation temperature falls.
- The lower evaporation temperature can be reached by decreasing pressure and the specific volume of the refrigerant will rise.

The capacity of a piston compressor is determined geometrically, the mass flow rate will fall and the capacity of the system decreases.

$$P_v = m_R \cdot (h_2 - h_1) \text{ (kW)}$$

Where:

$P_v$  – compressor output (kW)

$m_R$  – freon mass flow rate (kg/s)

$h_1$  – compressor inlet enthalpy (kJ/kg)

$h_2$  – compressor outlet enthalpy (kJ/kg)

The volume conveyed the piston compressor depend on evaporation and condensation pressure. The pressure ratio can be calculated as follows:

$$\Pi = P_c / P_o$$

Where:

$\Pi$  – Pressure ratio at compressor

$P_c$  – The pressure in the condenser

$P_o$  – The pressure in the evaporator

- The higher the volumetric efficiency at a particular pressure ratio, the higher the quality of the compressor.
- The capacity also falls as the condensation temperature rise.
- Piston compressor can operate effective, reliable in a particular range.

The clear operating range is limited by the usage limits:

- Maximum pressure ratio.
- Maximum possible final compression temperature.
- Minimum evaporation temperature.
- Maximum power consumption of compressor.



**Condenser** – task is to transmission the heat flows absorbed on the evaporator.

- The surface temperature of the condenser must be over the surrounding temperature or the cooling water temperature.

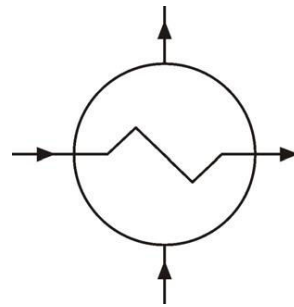


Fig. 11. Condenser symbol

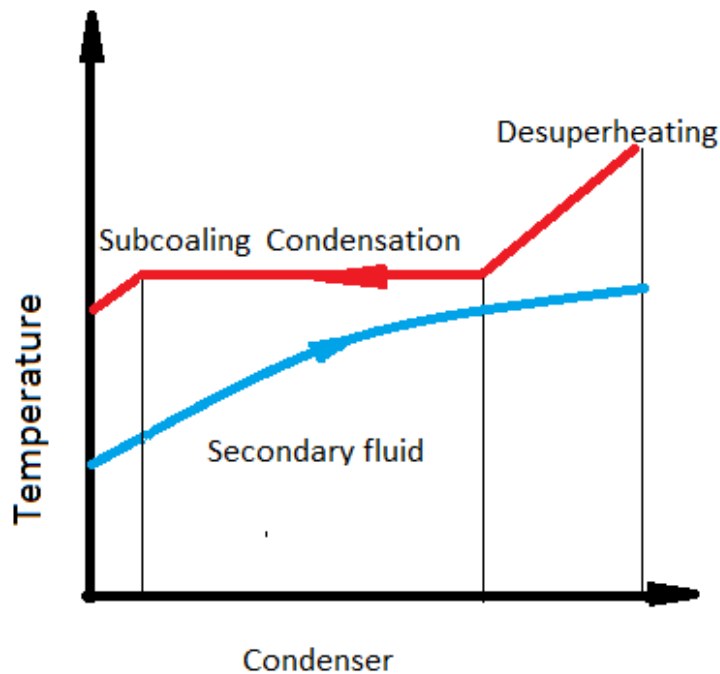


Fig.12. The condenser processes

- The refrigerant passes through three areas Fig.12.: desuperheating zone, condensation zone, supercooling zone.
- Heat desuperheating zone – superheated, gaseous freon cools at a constant pressure to the condensation temperature. The heat dissipation zone accounts for a 5% - 15% share of the total heat transmission of the condenser.

## Isenthalpic throttling

- Condensed refrigerant, condensation stage, must be supplied to the evaporator at low pressure and temperature.
- This happens during an isenthalpic change of state in the throttle.
- During process neither heat nor work is exchanged with the environment.

- Entropy increases take place during regulation as energy is dissipated inside the throttle during the process.

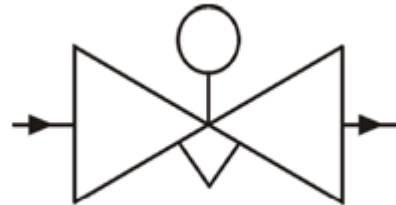


Fig. 13. Condenser symbol

- Proportion of the evaporated refrigerant during the throttling process can be identified from  $p,h$  diagrams. They contain lines that indicate the constant vapour content  $x$  in the wet steam area.
- This depends on the evaporation temperature and the supercooling of the refrigerant at the condenser.

## Isobaric evaporation

- The liquid refrigerant enters the evaporator takes heat energy from the surroundings and boils.

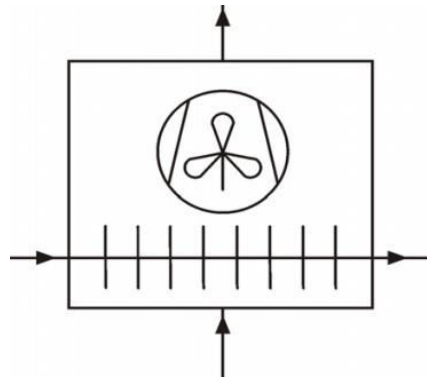


Fig. 14. Finned tube evaporator with axial fan symbol

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